

The LAT of mass "m" is mounted on springs with a combined spring constant "k". The springs + LAT have a resonant frequency $\omega_0 = \sqrt{k/m}$.

One end of the spring is connected to the aircraft (or the box wall). This end of the spring is driven by 1) Vibration and airpockets through contact with the airplane, or 2) Accoustics through pressure on the Transport Box wall.

1) Consider the airplane floor moving with some $x_{floor}(t)$ irrespective of the mass m. The floor acceleration has a fourier spectrum of $a_{floor}(\omega) = x_{floor}(\omega) \cdot \omega^2$. Assume the floor acceleration amplitude is independent of freq (as it would be for a narrow delta function acceleration of amplitude a_{floor_vib0}). Let $x_{lat}(t)$ be the position of the LAT mass m. The differential eqn for the system is:

$$\frac{d^2}{dt^2} x_{lat} + \gamma \frac{d}{dt} (x_{lat} - x_{floor}) + \frac{k}{m} (x_{lat} - x_{floor}) = a_0 \quad \omega_0^2 := \frac{k}{m}$$

$g := 9.8$ [m/sec^2] Acceleration of gravity

$a_{floor_vib0} := .5 \cdot g$ [m/sec2] Vibration acceleration amplitude

$\Delta x_{max} := \frac{2}{39}$ $\Delta x_{max} = 0.051$ [m] Max compressive travel of spring from loaded equilibrium

$\omega_0 := \sqrt{\frac{a_{floor_vib0}}{\Delta x_{max}}}$ $\frac{\omega_0}{2 \cdot \pi} = 1.556$ [Hz] Resonant freq of spring + LAT

$Q := .1$ Q of the spring (very low to suppress the resonance)

$\gamma := \frac{\omega_0}{Q}$ Damping constant

$x_{floor_vib}(\omega) := \frac{a_{floor_vib0}}{\omega^2}$ $x_{lat_vib}(\omega) := \frac{\omega_0^2 + i \cdot \gamma \cdot \omega}{\omega_0^2 - \omega^2 + i \cdot \gamma \cdot \omega} \cdot x_{floor_vib}(\omega)$ $a_{lat_vib}(\omega) := \omega^2 \cdot x_{lat_vib}(\omega)$

2) Consider a flat spectrum acoustic pressure wave acting on one side (the floor) of the box with force F(t). Assume the acoustic force on the wall is independent of frequency (db is the same at all frequencies)

$$\frac{d^2}{dt^2} x_{lat} + \gamma \frac{d}{dt} x_{lat} + \frac{k}{m} x_{lat} = \frac{F(t)}{m} \quad a_{floor_acoustic}(\omega) := \frac{F(\omega)}{m}$$

$db := 100$ [decibels] Sound pressure relative to 2×10^{-5} Newtons/m^2

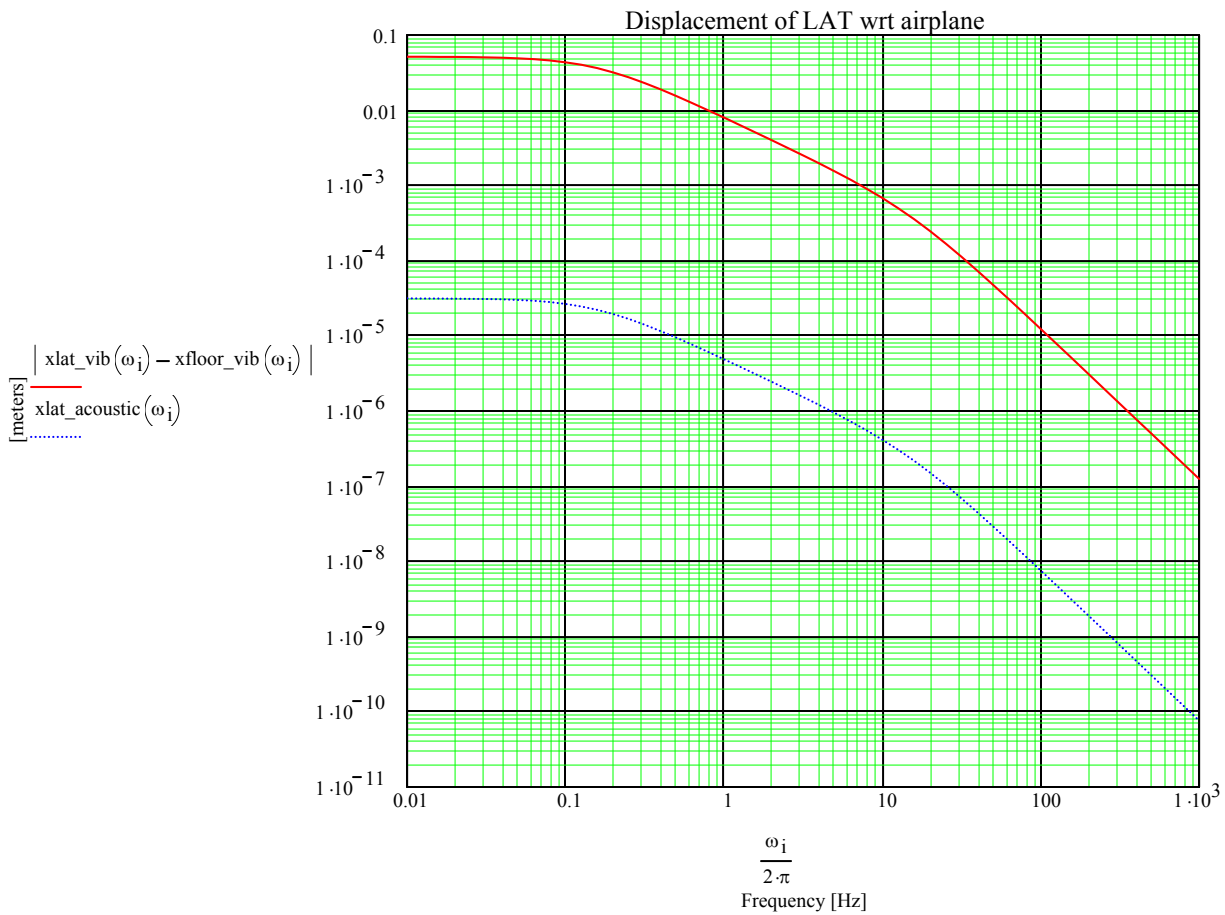
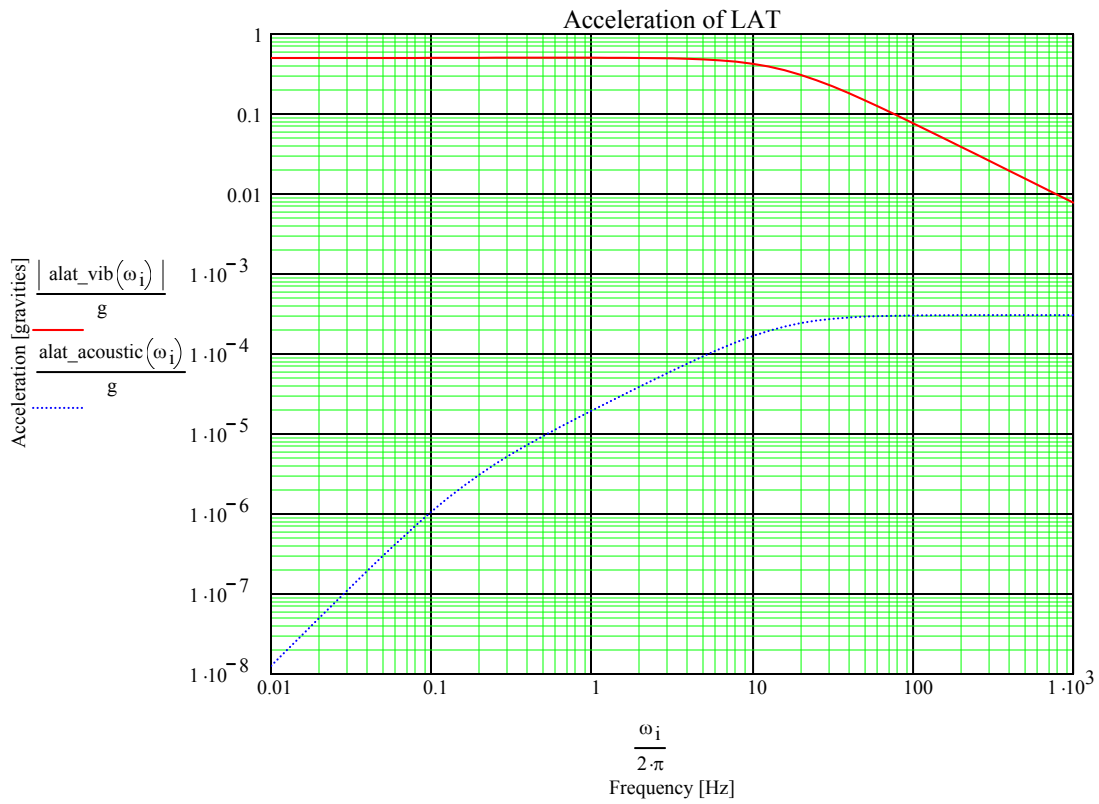
$A := 2.3$ [m^2] Area of side of box

$m := 4000$ [kg] Mass of LAT + perimeter ring + isolation frame

$a_{floor_acoustic0} := \frac{A \cdot 2 \cdot 10^{-5}}{m} \cdot 10^{\frac{db}{20}}$ [m/sec2] Acoustic acceleration amplitude

$a_{floor_acoustic0} = 3 \cdot 10^{-3}$

$x_{lat_acoustic}(\omega) := \frac{a_{floor_acoustic0}}{\sqrt{(\omega_0^2 - \omega^2)^2 + (\gamma \cdot \omega)^2}}$ $a_{lat_acoustic}(\omega) := \omega^2 \cdot x_{lat_acoustic}(\omega)$ $i := 0 .. 50$ $\omega_i := 2 \cdot \pi \cdot 10^{\frac{i}{10} - 2}$

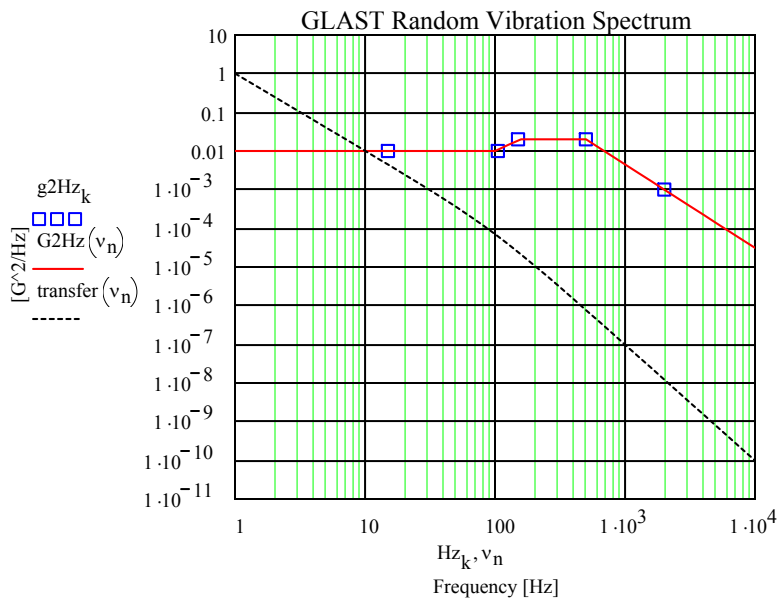


GLAST Random Vibration Spectrum (from Jim Haughton 202-767-4689 via Mike Lovellete) for Military transport aircraft.

$\text{Hz}_k :=$	$\text{g2Hz}_k :=$	$n := 0..40$	$k := 0..4$
15	.010	$v_n := 10^{\frac{n}{10}}$	
105	.010		
150	.020		
500	.020		
2000	.001		

$$\text{transfer}(\omega) := \left| \frac{1}{\omega^2} \cdot \frac{\omega^2 + i \cdot \gamma \cdot \omega}{\omega^2 - \omega^2 + i \cdot \gamma \cdot \omega} \right|$$

$$\text{G2Hz}(v) := 10^{\text{interp}(\log(\text{Hz}), \log(\text{g2Hz}), \log(v))}$$



$$\text{rms} := \sqrt{\int_{.1}^{10000} \text{G2Hz}(v) \cdot \text{transfer}(2 \cdot \pi \cdot v) \, dv} \quad \text{rms} = 0.05 \quad \text{[gravities]}$$