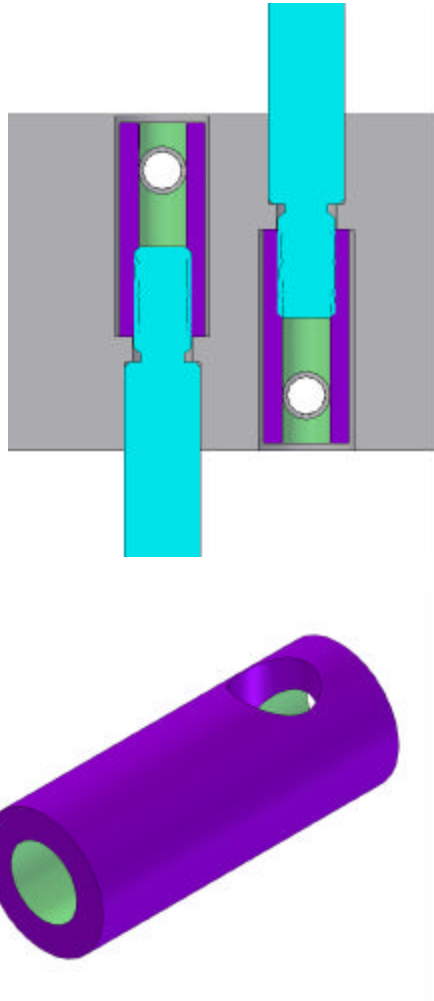


# Grid to Tracker Stud Interface Trade Study

Mike Menning

April 7, 2004

# Rear Insert



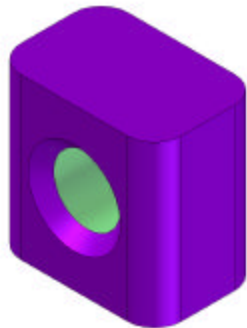
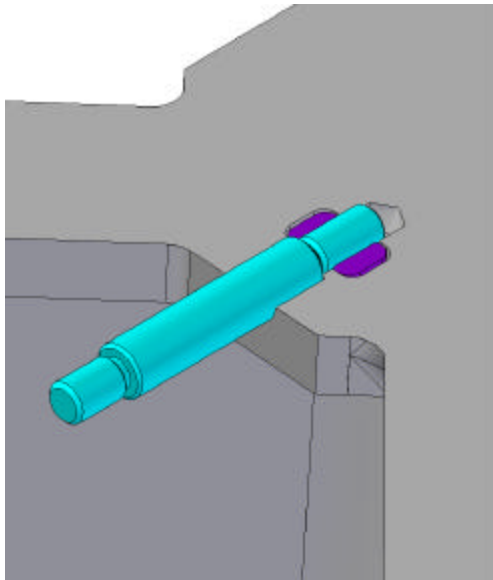
## Pro's

- No concentricity issues – wide range of float possible
- Serviceable at side flexure locations
- Eliminates grid stresses caused by bushing press fit
- Reduced tolerance requirements

## Con's

- Not applicable to corner flexures
- Requires constraint to retain insert in cavity and prevent insert rotation
- Creates four openings on each mid-side location, causing larger pin stresses

# Square Nut



## Pro's

- No concentricity issues – wide range of float possible
- Serviceable at corner flexure locations
- Eliminates grid stresses caused by bushing press fit
- Reduced tolerance requirements

## Con's

- Not serviceable at side flexure locations
- Requires shield from heatpipe bonding epoxy
- Requires constraint to retain insert in cavity and prevent insert rotation
- Creates four openings on each mid-side location, may increase stresses

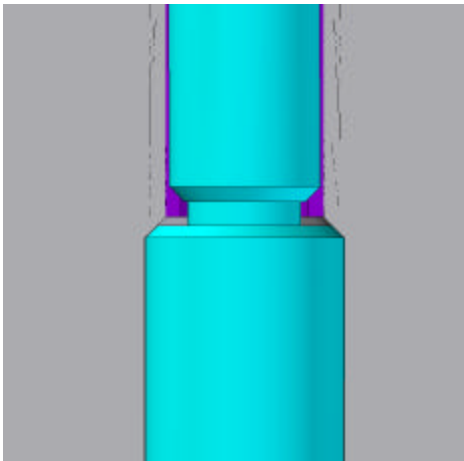
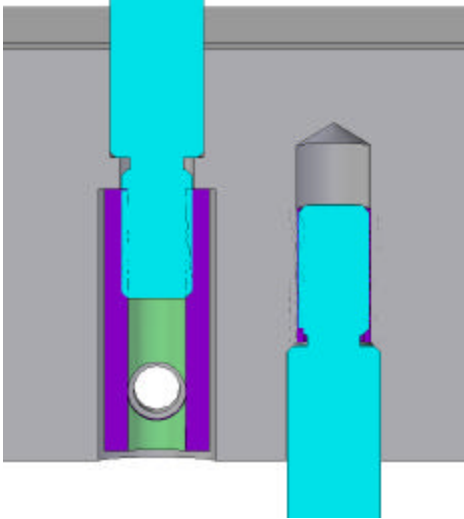
# Helicoil

## Pro's

- Simplest design
- Lowest cost design
- Applicable at both side and corner locations
- Eliminates grid stresses from the bushing press fit

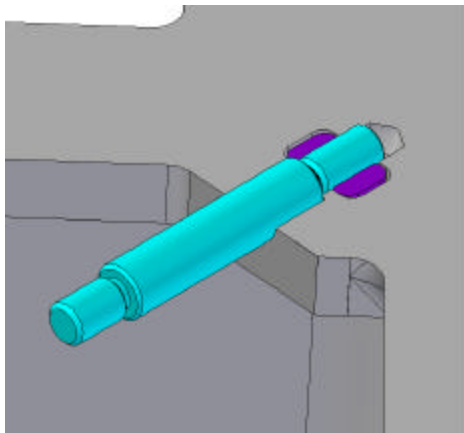
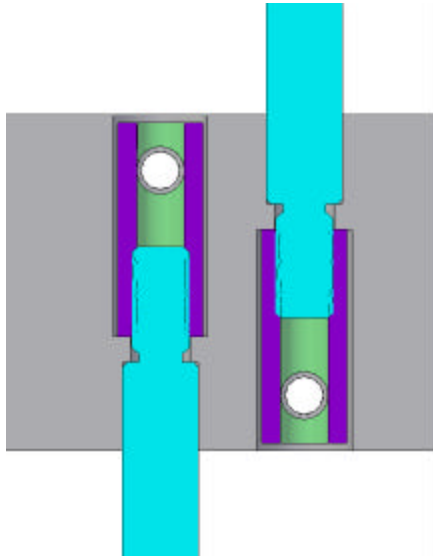
## Con's

- Helicoils are intended for secondary structure not primary structure
- May spin out during disassembly or during vibration environment
- Not readily repairable
- May not meet concentricity requirements
- Stud would seat on chamfer



# Recommendation

## Square Nut at Corners & Rear Insert at Sides



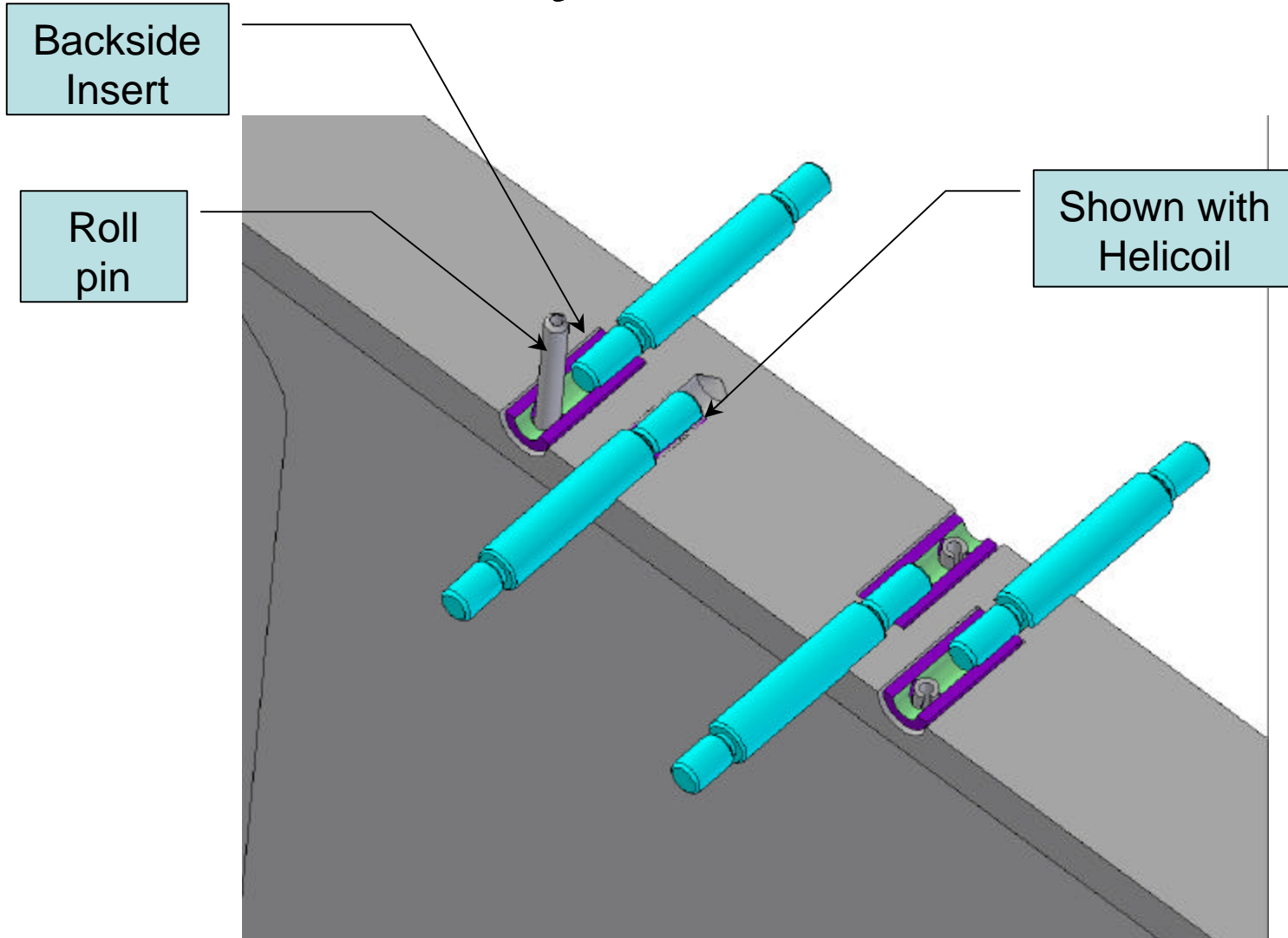
### Pro's

- No concentricity issues
- Serviceable at all locations
- Eliminates grid stresses from the bushing press fit
- Same stud design & load path for dual eccentric design

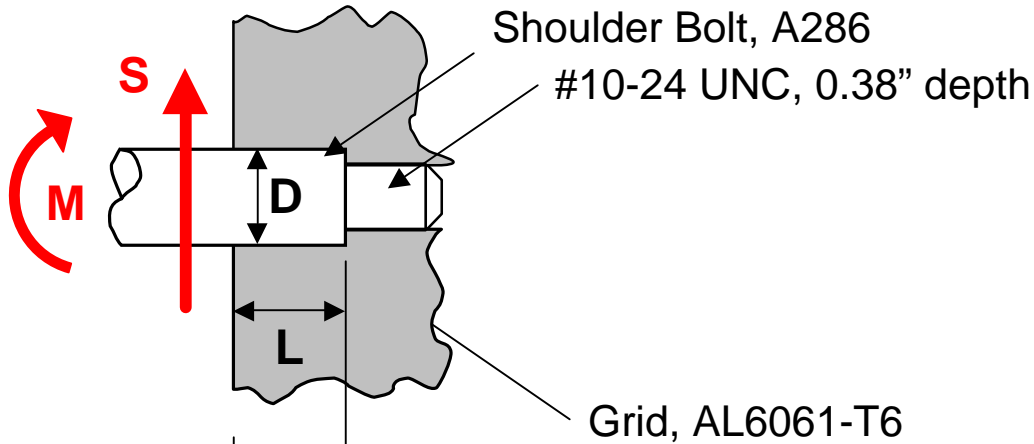
### Con's

- Constraint required to hold insert and square nut in position
- Additional Stress Analysis Required

# Cut-Away at Side Locations



# Shoulder Bolt Bearing Calculation Assumptions



| AL6061-T6 Bearing Strength |  |
|----------------------------|--|
| (e/D = 1.5):               | F <sub>by</sub> = 53 ksi; F <sub>bu</sub> = 69 ksi |
| (e/D = 2.0):               | F <sub>by</sub> = 61 ksi; F <sub>bu</sub> = 90 ksi |

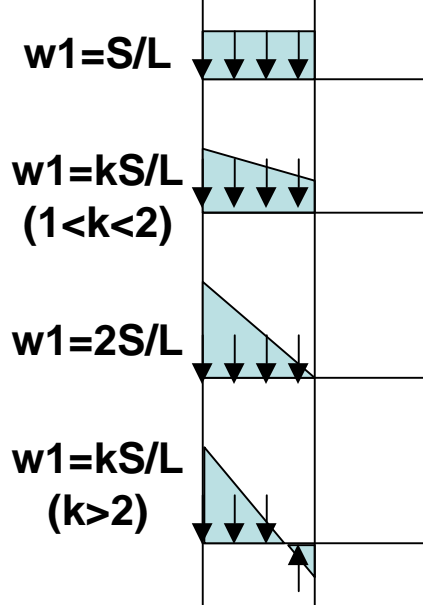
S<sub>max</sub> = 811 lbf = 3.61kN

M = 0 N-m

L = 0.279 in = 7.09 mm

D = 1/4 in = 6.35 mm

## Load distributions considered



Distribution 1 – Uniform through depth – **UNCONSERVATIVE**.

This distribution idealizes the bearing load of the pin on the hole, but is valid for short pins (L/D is a small number). In this case, L/D is 0.89, which is not small.

Distribution 2 – Trapezoidal; Linear varying through depth – **REALISTIC**.

This distribution is probably the most realistic case. However, there is not a clear method of calculating w<sub>1</sub> and w<sub>2</sub>.

Selected Method

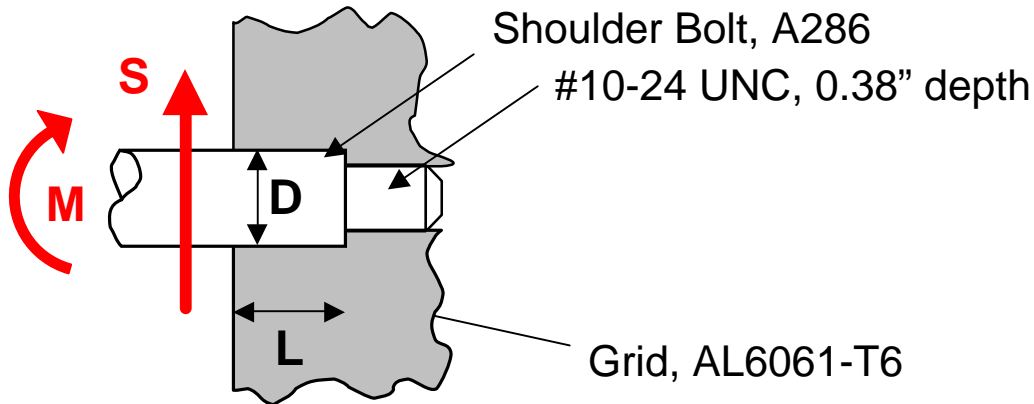
Distribution 3 – Triangular; Linear varying through depth – **CONSERVATIVE**.

This distribution represents the critical pin depth where less engagement would result in a trapezoidal distribution and more engagement would result in pin bending, i.e. "Distribution 4." This distribution is conservative and easy to calculate.

Distribution 4 – Bending; Linear varying through depth – **MOST CONSERVATIVE**.

This distribution is well suited for longer pins where pin bending may become a factor. In this case, it is not necessary to be overly conservative.

# Shoulder Bolt Bearing Calculation Margins



AL6061-T6 Bearing Strength  
 (e/D = 1.5): F<sub>by</sub> = 365 MPa; F<sub>bu</sub> = 476 MPa  
 (e/D = 2.0): F<sub>by</sub> = 421 MPa; F<sub>bu</sub> = 621 MPa

S<sub>max</sub> = 811 lbf = 3.61kN  
 M = 0 N-m  
 L = 0.279 in = 7.09 mm  
 D = 1/4 in = 6.35 mm

## Bearing Stress in Alum Grid Using Load Distribution 3

Pin engagement into Grid is

L = 7.09 mm (0.279 in)  
 D = 6.35 mm (1/4 in)

Bearing load using load distribution 3

$$w_1 = \frac{2 \cdot S}{L} = 1.018 \frac{kN}{mm}$$

Bearing Stress calculation

$$s_{br} = \frac{w_1}{D} = 160MPa \longrightarrow \begin{matrix} MS_u = 1.13 \\ MS_y = 0.83 \end{matrix}$$

## Margins using Shear out-bearing strength from Bruhn (more realistic; based on empirical data)

Calculate Edge distance (e/D)

e = 7.493 mm (0.295 in); D = 6.35 mm (1/4 in)  
 e/D = 1.18

$$P_{Bru} = K_{Bru} F_{tu} A_{Br}; P_{Bry} = K_{Bry} F_{ty} A_{Br} \quad \text{AL6061-T6}$$

$$K_{Bru} = 1.2; K_{Bry} = 1.1 \quad \text{Ftu}=296MPa$$

$$F_{tu} = 296MPa; F_{ty} = 262MPa \quad \text{Fty}=262MPa$$

$$A_{Br} = (7.09mm)(6.35mm) = 45.02 \cdot 10^{-6} mm^2$$

$$P_{Bru} = (1.2)(296MPa)(45.02 \cdot 10^{-6} mm^2) = 15991N$$

$$P_{Bry} = (1.1)(262MPa)(45.02 \cdot 10^{-6} mm^2) = 12975N$$

$$\text{Adjusted Margin calculation} \longrightarrow \begin{matrix} MS_u = 2.16 \\ MS_y = 1.88 \end{matrix}$$